## Bond Graph for Modelling, Analysis, Control Design, Fault Diagnosis

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## Bond Graph Research Group

Laboratoire d'Automatique et d'Informatique Industrielle de Lille *Ecole Centrale de Lille* 

- 6 academics (2 Profs, 2 associate profs, 2 assistant profs)
- 10 PhD students

Application areas: power systems (electrical machines, photovoltaic systems, fuel cells), thermofluid process, car industry

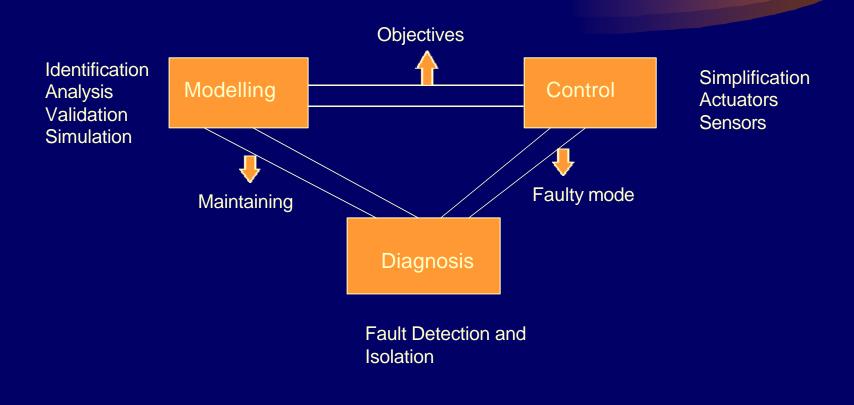
## Studies performed in collaboration with Peugeot -Citroën

- Mechatronic design of an automatic gear box
- Clutch management and drive comfort
- Mechatronic design of an active hydraulic suspension
- Thermal comfort regulation in a car interior
- Modelling and simulation of a fuel cell system
- Analysis of structural properties of bond graph models
- Robustness of control laws for systems with parametric uncertainties
- •

## Why a bond graph approach?

- Multidisciplinary systems  $\rightarrow$  need for a communication language between people from different physical domains
- Need for models with physical insight (virtual testing facility)
- Unified modelling methodology for knowledge storage in model libraries
- Integrated (« mechatronic ») design of controlled systems

## Mechatronic design

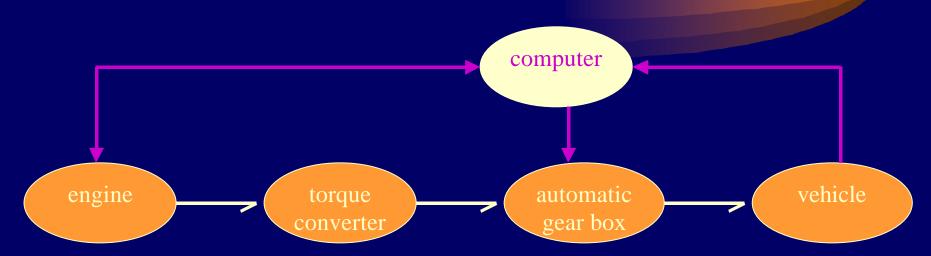


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Rosario - 22/11/02

# 1 - Mechatronic design of an automatic gear box



## Complete driveline

Differential, Transmission shaft Wheels, Vehicle mass, air resistance, road friction

## 1 - Mechatronic design of an automatic gear box Problem statement

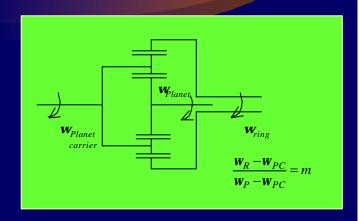
- Design control laws for the driving of an automatic gear box by a computer with the following objectives:
  - Complete satisfaction of the customer corresponding to a variation of the engine torque as continue as possible
     (no jerk in acceleration during a shift)
  - Respect of technological constraints (actuator response duration, minimization of the energy dissipated in the clutchs)

When to shift? How to shift?

## 1 - Mechatronic design of an automatic gear box Automatic gear box : physical scheme

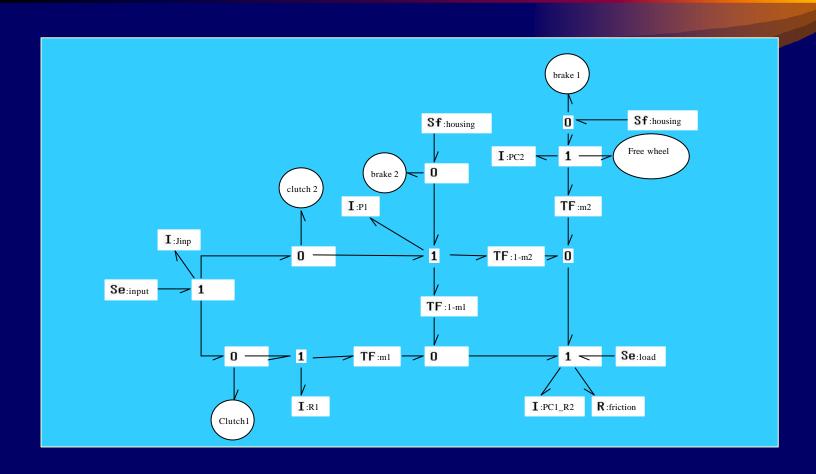
Arrangement of 2 epicyclic gear trains which allows 3 ratios plus one reverse

Different ratios : one element blocked or 2 elements maintained at the same speed

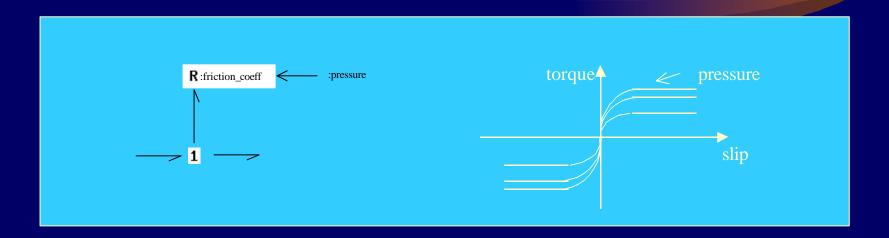


Clutches between 2 rotating elements
Brakes between one element and the housing block

# 1 - Mechatronic design of an automatic gear boxBond graph model of the automatic gear box



# 1 - Mechatronic design of an automatic gear boxBond graph model of a clutch or a brake



Coulomb friction depending on the pressure applied on the clutch disks, defining the « limited torque »

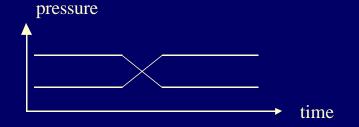
If clutch torque < limited torque → clutch closed, all the torque transmitted If clutch torque = limited torque → clutch opened, slipping velocity

## 1 - Mechatronic design of an automatic gear boxDecision block

Contains the schift schedule (diagram throttle position vs vehicle speed) which permits to know « when to shift »

Different programs: economical, sport, snow

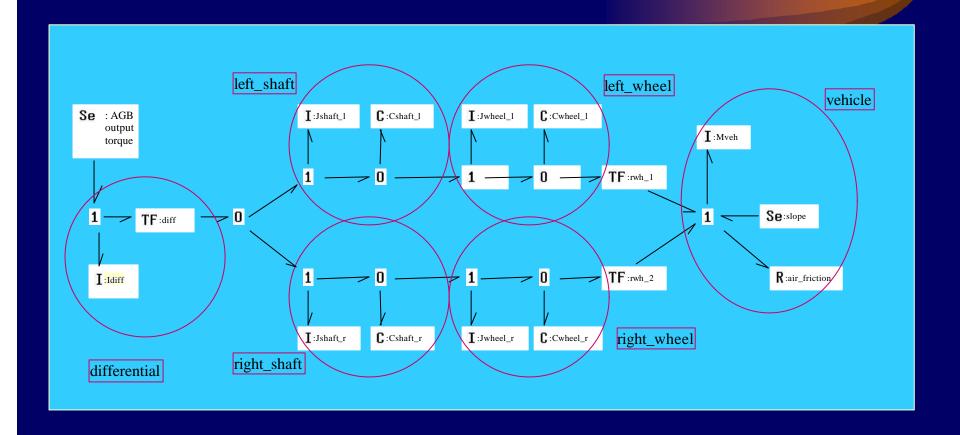
When a shift is decided, the different pressures in the clutches are controlled



#### « How to shift »:

- action on only 2 clutches or 2 brakes at the same time,
- control of the pressure to have a smooth shift and no jerk in acceleration

# 1 - Mechatronic design of an automatic gear boxBond graph model of the vehicle



## 2- Clutch management and drive comfort

#### Ojectives:

- Reduce the well-known fore and aft oscillation of a vehicle occurring when a sudden torque variation takes place in the transmission (throttle step sollicitation)
- Satisfy comfort and driving pleasure

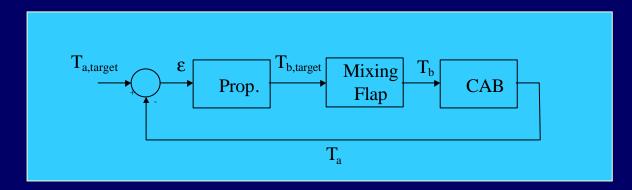
#### Means:

- Define an hydraulic-electronic-mechanical actuator transforming the numerical output into pressure on the plates of the clutch
- Design control laws for this electrohydraulic servovalve

# 3 - Thermal comfort regulation in a car interior

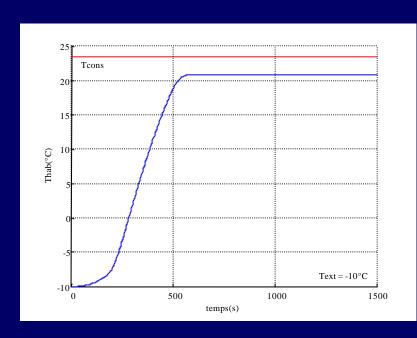
#### <u>Usual climate control</u>

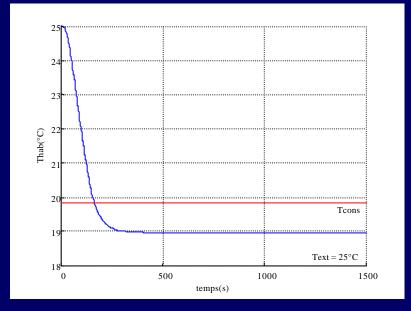
- Try to reach and maintain the passenger compartment temperature to a specified target temperature.
- → The regulator acts on the mixing flap to increase or decrease the blown air temperature. Usually, a proportional strategy is used to control the mixing flap



## 3 - Thermal comfort regulation in a car interior Usual climate control

! Usually the air temperature in the compartment does not reach the target temperature.





heating

cooling

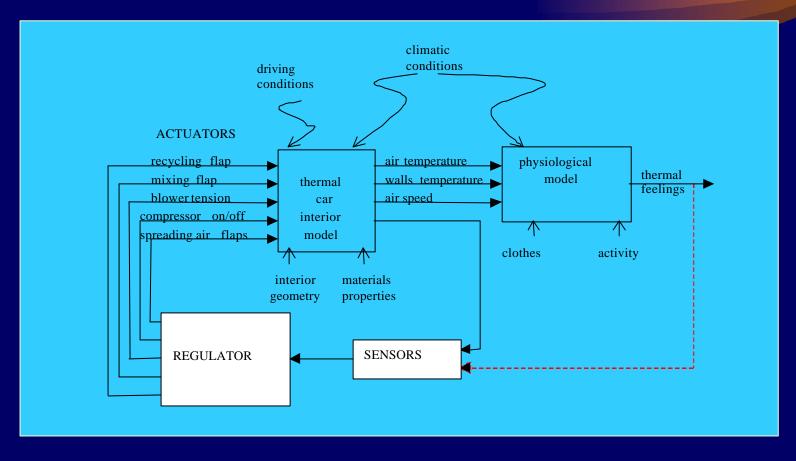
## 3 - Thermal comfort regulation in a car interior Comfort strategy

- comfort: much more than only thermal comfort. Our five senses, our cerebral state, our thermal state have an influence on our comfort estimating.
- thermal sensations rather than thermal comfort a very subjective notion.
- in PSA Peugeot-Citroën, quantitative scale to evaluate a thermal sensation: an integer between 1 (very cold) and 9 (very hot) sensation.

#### Objectives:

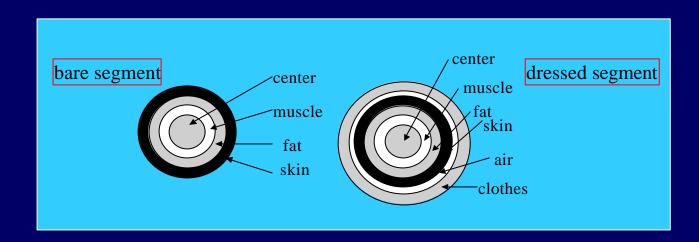
define a regulation strategy for a climate controller for car interior, taking into account the car passenger's thermal sensations.

## 3 - Thermal comfort regulation in a car interior Block representation of the model

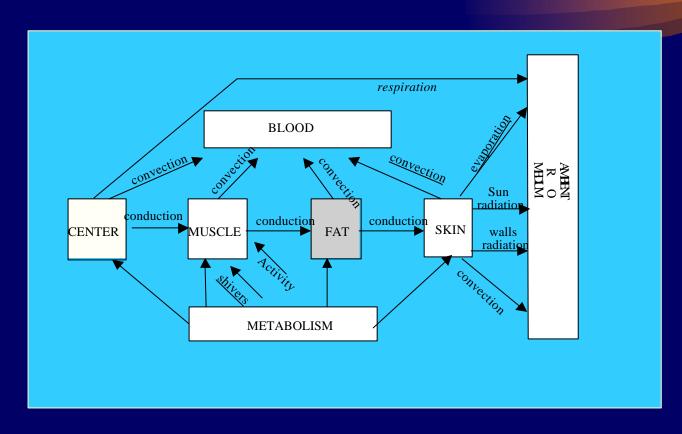


## 3 - Thermal comfort regulation in a car interior Physiological model

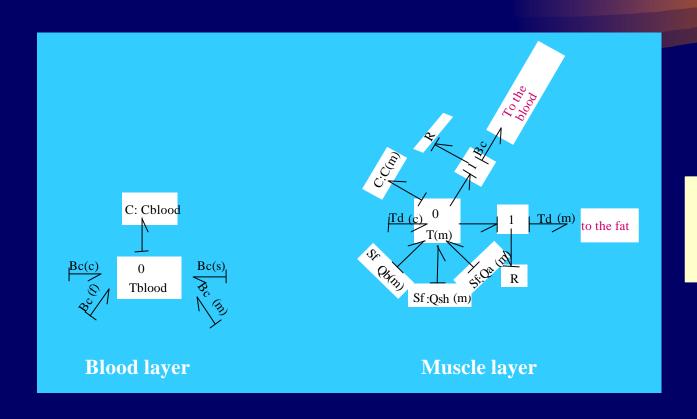
- → human body divided into seven parts called **segments**: the head, the trunk, the left arm, the right arm, the hands, the legs and the feet.
- → head and hands segments are bare.



## 3 - Thermal comfort regulation in a car interior Physiological model



## 3 - Thermal comfort regulation in a car interior Physiological model



For each segment

## 3 - Thermal comfort regulation in a car interior Physiological mathematical model

- x state vector (order = 57): temperature, water mass, sudation production and water partial pressure of the layers.
- u input vector (size = 14): air temperature and air speed of the ambient air close to the 7 segments.
- d disturbance vector (size = 35): ambient air humidity, sun and wall radiation on clothes and skin layers.
- y output vector (size = 7): thermal feelings of the 7 segments.

$$\begin{cases} \dot{x} = f(x, u, d) \\ y = g(x, u, d) \end{cases}$$

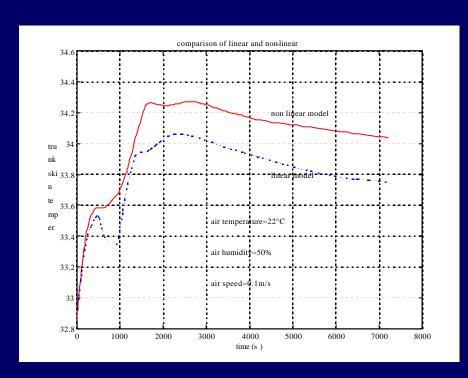
## 3 - Thermal comfort regulation in a car interior Linearized physiological mathematical model

- A nominal functioning point defined as a comfortable situation for the human (air temperature=299K, air speed=0.5m/s and humidity=50%).
- Two linear models containing saturations, because heat transfers are different whether the body is warm or cold.

$$\begin{cases} \dot{x} = (bA_c + \overline{b}A_w)x + B \cdot u + E \cdot d + F \\ y = C \cdot x + D \cdot u \\ m < K \cdot x < M \end{cases}$$

- **b** = 1 if the body is cold, and 0 if not.
- F vector that results from constant thresholds due to saturations.
- **K** matrix selecting the x components concerning the saturations.

## 3 - Thermal comfort regulation in a car interior Physiological model: simplifications





Trunk skin temperature

Trunk sensations

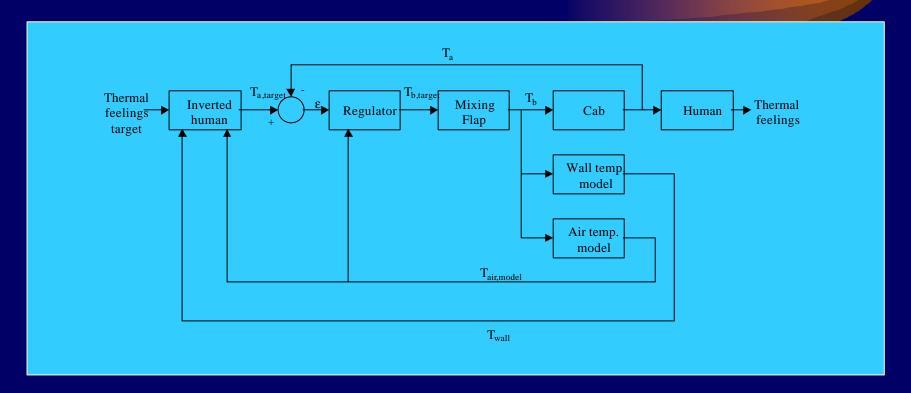
## 3 - Thermal comfort regulation in a car interior Proposed comfort strategy

#### Control strategy based on the thermal feelings:

- Determine the air temperature close to the head driver to ensure him a comfortable thermal feeling (level 5 in PSA scale)
- Take into account the wall temperatures and the air flows in the compartment.
- Compute the best air temperature target for the driver to be comfortable by using the inverted human model

In case of chilly driver, a target sensation superior to 5 can be asked for

## 3 - Thermal comfort regulation in a car interior Proposed comfort strategy



Predictive control (GPC)

## 3 - Thermal comfort regulation in a car interior Comfort strategy - Air temperature model

linear convex formulation:

$$T_a = (\frac{a}{1+t_1s} + \frac{1-a}{1+t_2s})(l_aT_{out} + (1-l_a)T_b)$$

- $T_a$  air temperature,
- $T_{out}$  outside air temperature,
- $T_b$  blown air temperature,
- convex parameter depending essentially of the blown air flow and the vehicle speed.

There are two dynamic modes: a fast mode caused by the mass transfer and a slow mode caused by the thermal transfers with the outside.

## 3 - Thermal comfort regulation in a car interior Comfort strategy - Wall temperature model

first order model:

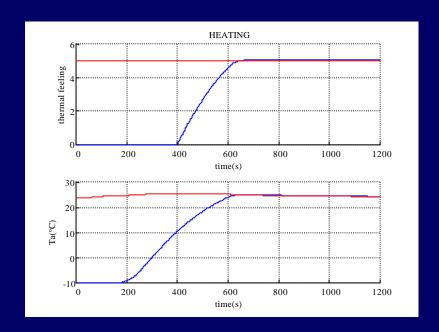
$$T_w = (\frac{1}{1 + \boldsymbol{t}_w s})(\boldsymbol{l}_w T_{out} + (1 - \boldsymbol{l}_w) T_a)$$

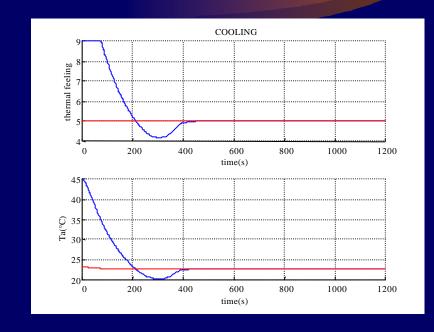
- $T_w$  the surface temperature,
- $T_a$  the air compartment temperature close to the surface.

Experiments in a wind tunnel to identify the parameters of each air temperature model and wall temperature model:

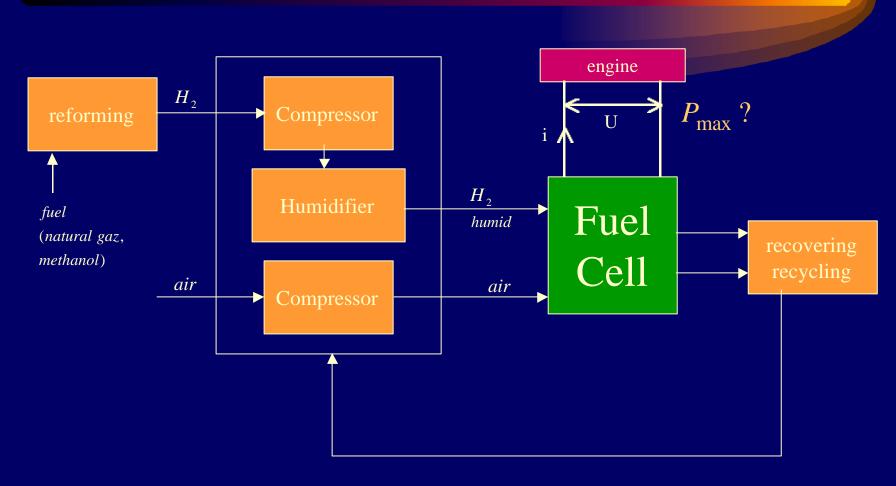
several wind-tunnel air flows (for the air flow due to the vehicle speed), several outside temperatures, several blown air temperatures and flows

## 3 - Thermal comfort regulation in a car interior Comfort strategy



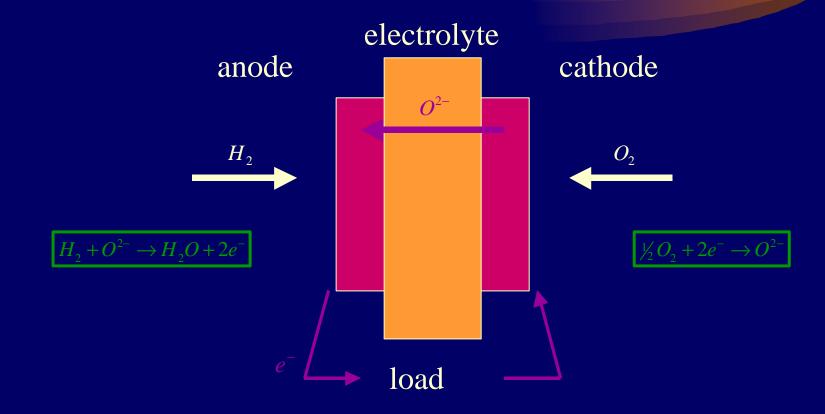


## 4 – Modelling of a fuel cell system



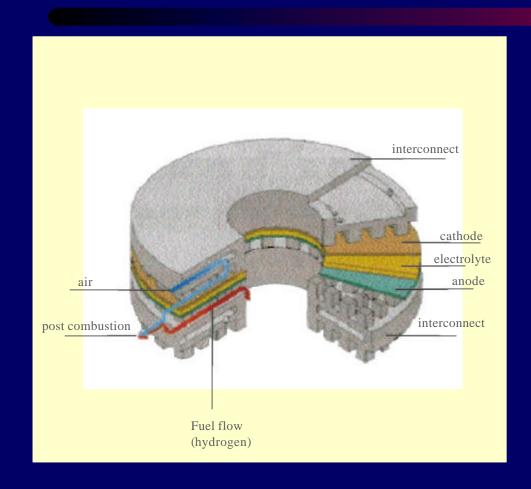
## 4 – Modelling of a fuel cell

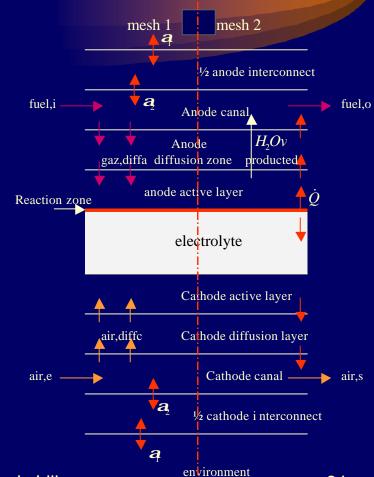
Fuel cell: principle



### 4 – Modelling of a fuel cell

### Fuel cell: tubular design





### 4 – Modelling of a fuel cell Variables

Pressure: P

Fluid

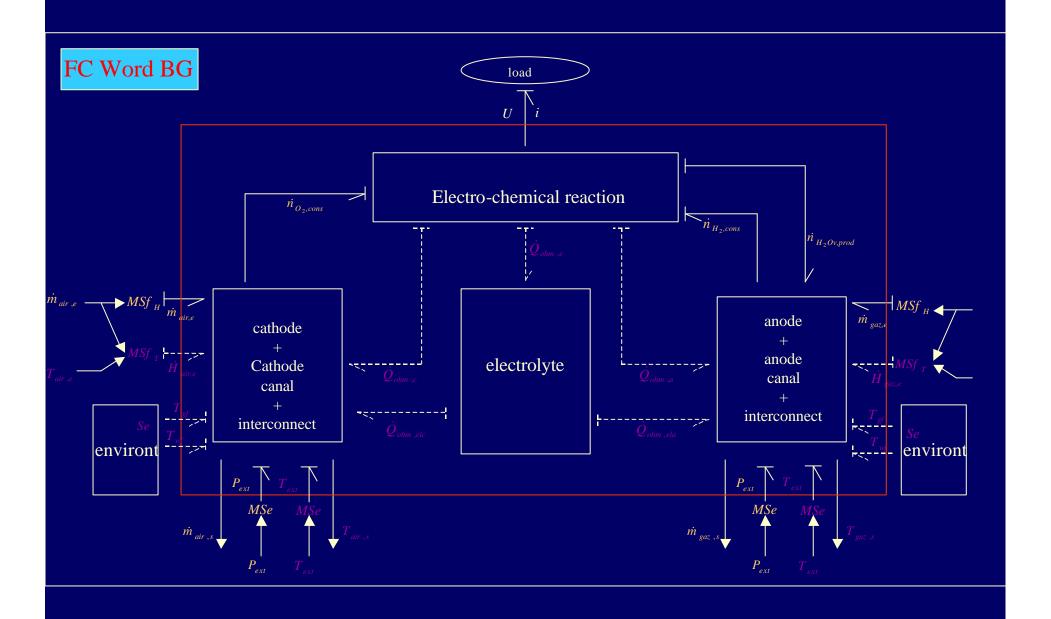
- Temperature: T
- Mass or molar flow rate: m or n

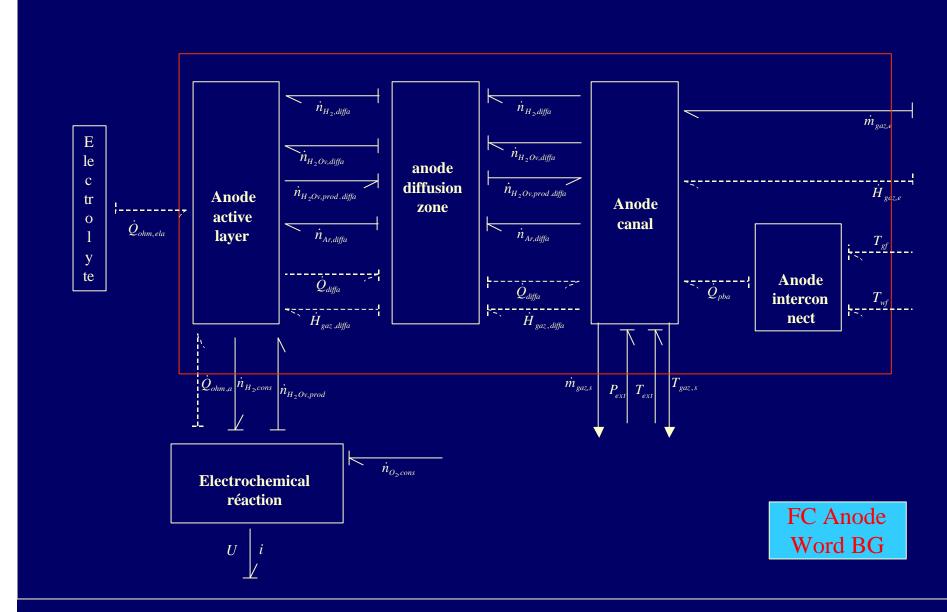
Chemical • Molar flow rate: *n* 

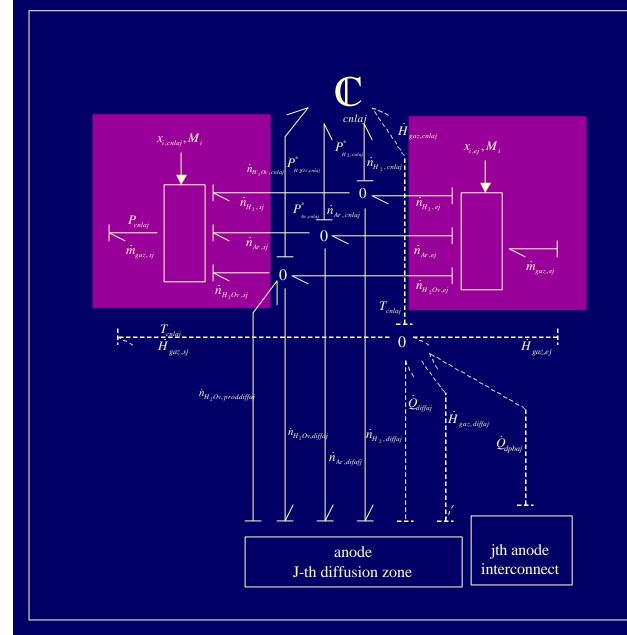
reaction • Chemical potential: m

Electrical

- Current: i
- Voltage: U





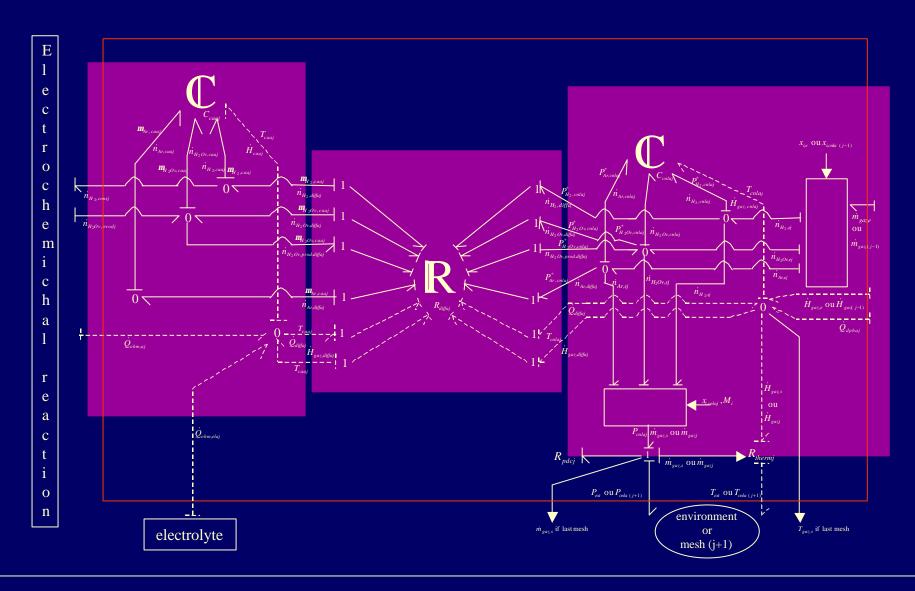


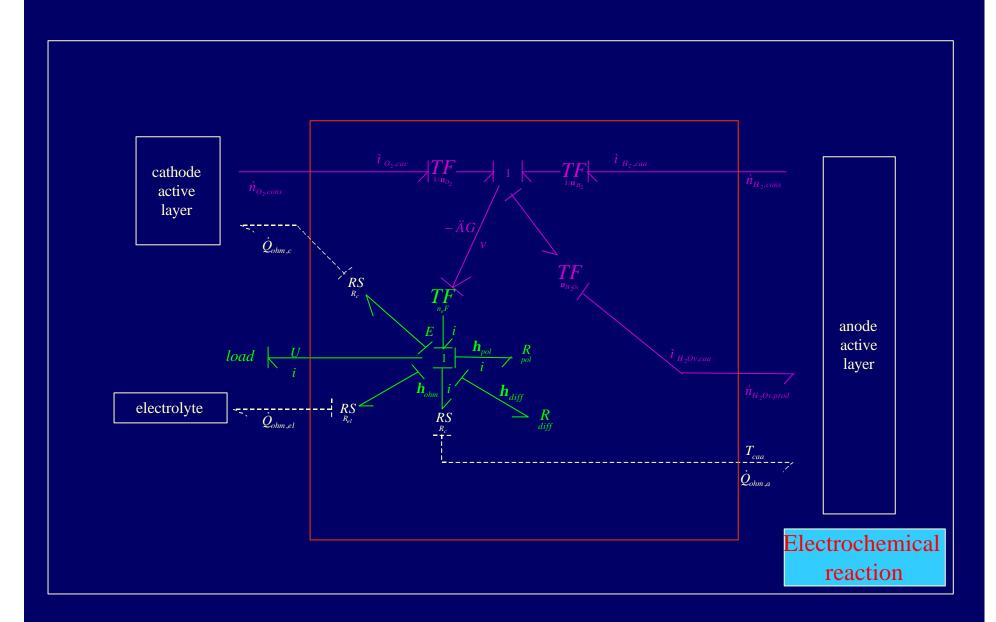
Fuel supply

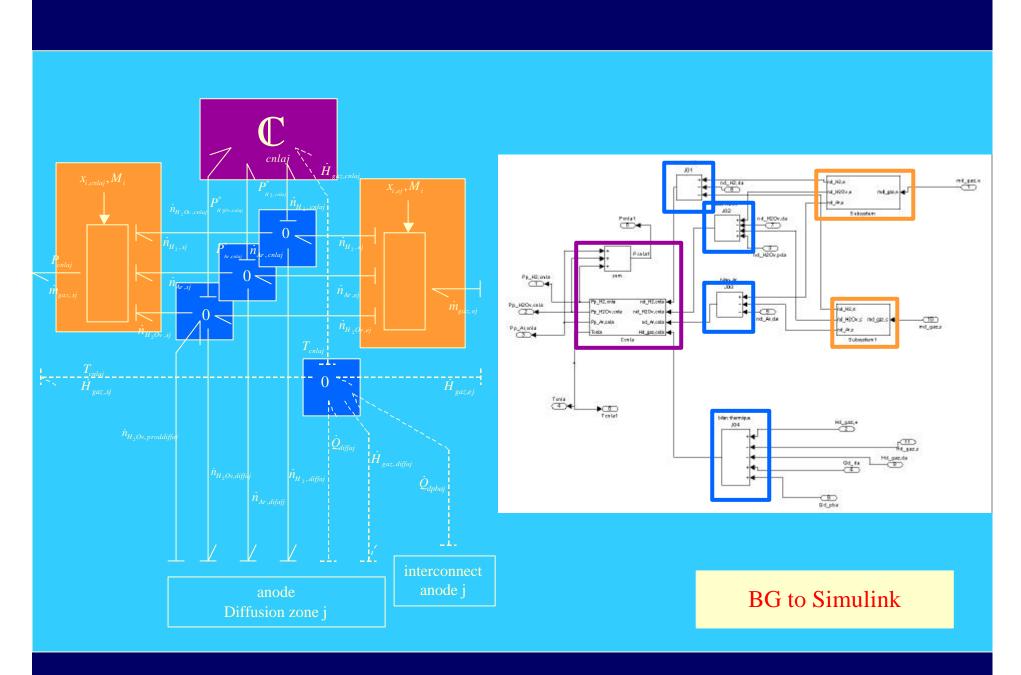
$$\dot{m}_{_{gaz,ej}}$$

$$\dot{n}_{H_2,ej} = \dot{m}_{gaz,ej}. x_{H_2,ej}. M_{H_2}$$

Anode canal BG

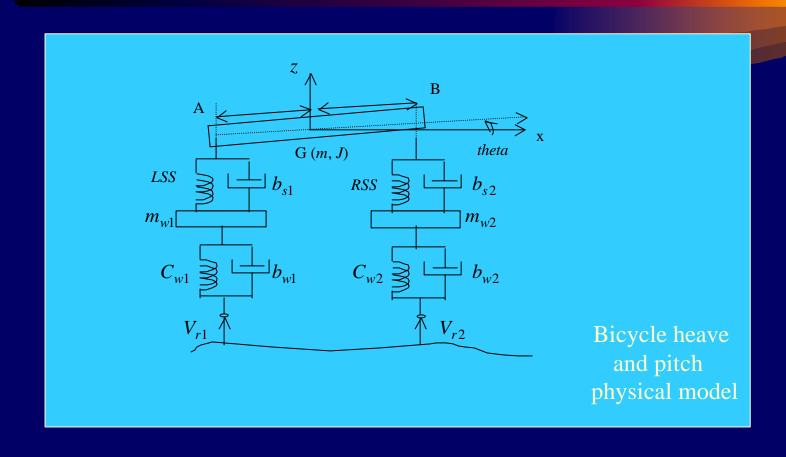


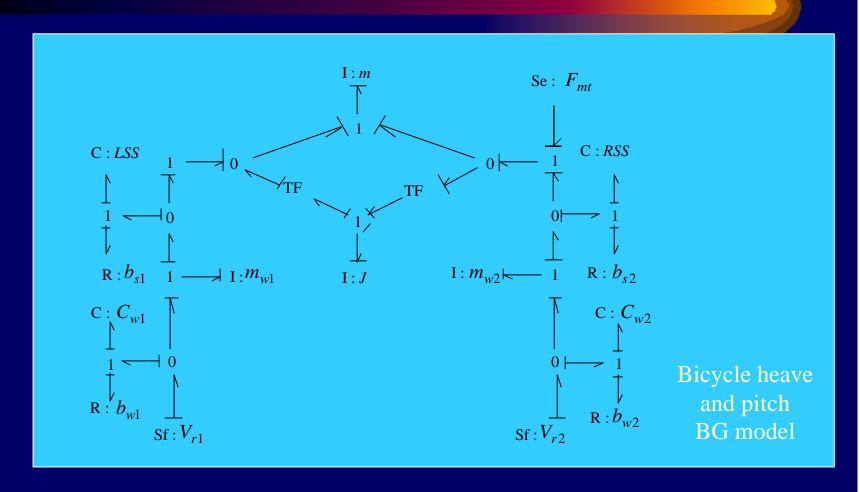




#### 4 – Modelling of a fuel cell Results

- → Complete dynamic model of the fuel cell system (no similar result in the literature, only static models)
- → Simulation results validated by comparison with experimental data
- → Work with PSA is running for:
  - → Control designing : how to maximize the power delivered by the fuel cell system
  - → Fault diagnosis





## 5 - Structural properties of bond graph models Passive model

#### • State equation

$$\dot{x} = Ax + Ed$$

$$x = \begin{pmatrix} p_I \\ q_C \end{pmatrix} = \begin{pmatrix} \text{inertia impulses} \\ \text{spring displacements} \end{pmatrix}$$

$$d = \begin{bmatrix} d_1 \\ d_2 \end{bmatrix} \qquad d_1 = \begin{bmatrix} F_{masstransfer} \end{bmatrix} \qquad d_2 = \begin{bmatrix} V_{road1} \\ V_{road2} \end{bmatrix}$$

## 5 - Structural properties of bond graph models Passive model

• State equation

$$\dot{x} = Ax + Ed$$

- <u>order *n* of a model</u>: number of I and C elements in integral causality when a preferred integral causality is assigned to the bond graph model
- BG-rank *q* of the state space matrix *A*: number of I and C elements in derivative causality when a preferred derivative causality is assigned to the bond graph model.
- <u>number of structurally null modes of A-matrix</u>: number of I and C elements which have to stay in integral causality when a preferred derivative causality is assigned to the bond graph model

## 5 - Structural properties of bond graph models Passive model

- State equation  $\dot{x} = Ax + Ed$
- minimum number of actuators for the model to be controllable :
  - If BG-rank A = n, the model is controllable with a <u>single actuator</u>
  - If BG-rank A = n-k, for the model to be controllable, k well-located actuators are needed
- minimum number of sensors for the model to be observable:
  - idem

# 5 - Structural properties of bond graph models Design of the measurement and control architecture for the active system

- <u>definition of the control objectives</u>
  - what variables to be controlled?
  - for what performances (dynamical or frequential criteria)?
  - with what strategy (pole placement, disturbance rejection, ...)?
- what type of control law?
  - state feedback?
    - Is the state measurable?
  - output feedback?

## Design of the measurement and control architecture for the active system

#### **Choice here**

State feedback for pole placement and rejection of the disturbance corresponding to the mass transfer due to driver actions (braking or accelerating) on the 2 velocity variables (heave and pitch)

- ! the 2 variables (absolute velocities) to be controlled are not measurable
  - → an observer is needed
- ! We want to perform input/output decoupling
  - → 2 control inputs are needed

#### Design of the measurement and control architecture for the active system

2 outputs to be controlled: not measurable  $y = \begin{vmatrix} y_1 \\ y_2 \end{vmatrix} = \begin{vmatrix} v_m \\ w_I \end{vmatrix}$ 

$$y = \begin{bmatrix} y_1 \\ y_2 \end{bmatrix} = \begin{bmatrix} V_m \\ \mathbf{w}_J \end{bmatrix}$$

(Df\*)

 $\Rightarrow \text{ measurement vector } z = \begin{bmatrix} z_1 \\ z_2 \end{bmatrix} = \begin{bmatrix} V_{rel1} \\ V_{rel2} \end{bmatrix}$ 

(Df)

⇒ 2 control inputs 
$$u = \begin{bmatrix} u_1 \\ u_2 \end{bmatrix} = \begin{bmatrix} F_{act1} \\ F_{act2} \end{bmatrix}$$
⇒ disturbance vector

(MSe)

measurable to be rejected  $d_1 = \begin{bmatrix} F_{masstransfer} \end{bmatrix}$ 

$$d_1 = \left[ F_{masstransfer} \right]$$

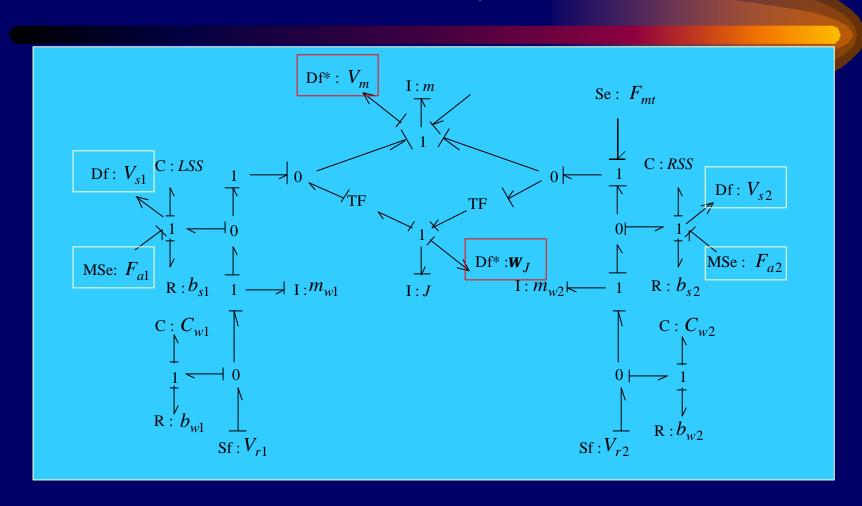
(Se)

non- measurable

$$d_2 = \begin{bmatrix} V_{road1} \\ V_{road2} \end{bmatrix}$$

(Sf)

## Design of the measurement and control architecture for the active system



#### Conclusion

- \* bond graph: language quite « strange » which needs a learning time
- \* Could appear difficult to implement, but what is difficult is PHYSICS
- \* more and more introduced in the industrial world in France (better than in the academic world!)